

AMENDMENTS TO THE SPECIFICATION

Please substitute the following amended paragraph for the second full paragraph on page 3 of the specification as originally filed:

~~---Claim 1 of the~~ The present invention defines a swash plate type variable capacity fluid machine for supplying and discharging applied fluid comprising: a conical body and a disk body rotatably supported with their center axes crossing, the conical body and the disk body confronting each other; an enclosure wall whose inner spherical surface surrounds a space in front of a circular disk surface of the disk body, the spherical surface being concentric with the disk surface; partitioning means for dividing the space between the conical body and the disk body into a plurality of variable capacity compartments in respect of radius lines on the disk surface; and supplying/discharging through holes communicating with the variable capacity compartments; characterized in that: the partitioning means comprises a partition plate movably fitted in a diameter groove of the conical body and an abutment line formed between the conical body and the disk body on their confronting surface; the enclosure wall is integrally connected to the disk body; and the conical body and the disk body are provided with a synchronous mechanism thereby synchronizing their rotation about their center axes.---

Please substitute the following amended paragraph for the third full paragraph on page 3 of the specification as originally filed:

~~---Claim 2 of~~ Furthermore, the present invention defines the swash plate type variable capacity fluid machine ~~according to claim 1~~, wherein the conical body has a rear axle integrally extending along its center axis on the rear side, the rear axle having an end surface onto which an increased pressure is delivered from the variable capacity compartments via pressure channels, the end surface thus applying a counter force in the direction of the variable capacity compartments.---

Please substitute the following amended paragraph for the fourth full paragraph on page 3 of the specification as originally filed:

~~---Claim 3 of~~ Furthermore, the present invention defines the swash plate type variable capacity fluid machine ~~according to claim 2~~, wherein the rear axle has a cylindrical axle integrally constructed to support the rear axle, the cylindrical axle having a plurality of through holes made on its entire circumference at regular intervals, thereby permitting applied fluid to pass through the through holes.---

Please substitute the following amended paragraph for the paragraph bridging pages 3-4 of the specification as originally filed:

~~---Claim 4 of Furthermore, the present invention defines the swash plate type variable capacity fluid machine according to claim 1, wherein the disk body has supplying/discharging through holes communicating with the variable capacity compartments on one end and with a gate member on the other end, the gate member gating supplying/discharging channels in response to its predetermined angular positions, thereby supplying and discharging applied fluid.---~~

Please delete the second full paragraph on page 13 of the specification:

~~The enclosure wall 65a is screwed in the disk 5. One of its open ends is supported by a flat bearing 66 fixed by a spring pin 66a. The flat bearing 66 has an axial groove (not shown) on its sliding surface 66b for the purpose of lubrication. The axial groove and the spring pin 66a have a hollow space therein, which acts as a pressure channel to thereby receive discharging pressure for lubrication and pressurization of the axial end. A cone axis 71 has a plurality of shallow orbit grooves on its circumference in order to support a cross-axis supporting member 77 with lubrication. A ball seat 74 is provided with a sealing means such as an O-ring 245 on its large diameter, thereby obtaining a force from fluid received on an outer end surface 74a without a need for a return flow channel leading to a tank.~~

Please substitute the following amended paragraph for the second full paragraph on page 21 of the specification as originally filed:

---In operation, the variable capacity compartments are defined within the spherical enclosure wall by the partitioning means, and a relative displacement of the compartments causes fluid to be supplied and discharged, or the supplying and discharging of fluid cause the compartments to be relatively displaced. Then, the disk body and enclosure wall integrally constructed rotate along with the conical body and the partition plate in synchronization by the synchronous mechanism, thereby reducing the relative speed between the enclosure wall and the partition plate. Consequently, the partition plate can be improved in its fluid tightness and durability. With this claimed structure-(~~claim 1~~), the rotation of the conical body is limited within the hemispherical space facing the disk body in the enclosure wall, and therefore, the fluid machine can be easily constructed with a simple hemispherical structure of the enclosure wall.---

Please substitute the following amended paragraph for the third full paragraph on page 21 of the specification as originally filed:

---In the structure with the conical body having the rear axle extending along its center axis on the rear side, and its end surface forming the pressure-exposed area, the increased pressure from the variable capacity compartments is applied onto the end surface of the rear axle via pressure channels to act as the counter force against the

pressure from the variable capacity compartments, thereby preventing the conical body from leaving apart from the confronting disk body. ~~(claim 2).~~ Further ~~to the advantages described in claim 1~~ advantageously, this arrangement allows improvement of durability and expansion of the applicable fluid thanks to the modified sliding structure, at the same time effectively keeping both the conical and disk bodies abut on each other along the radial abutment line in fluid-tight condition.

Please substitute the following amended paragraph for the paragraph bridging pages 21-22 of the specification as originally filed:

---With the structure that the end surface of the rear axle is supported by the cylindrical axle provided with the through holes, the fluid is permitted to radially eject from the through holes making the cylindrical axle self-centered with its centripetal force, thereby sustaining the rear axle about the center axis. The ejecting fluid that lubricates and supports the rear axle further has a cooling effect of the rear axle on its all circumference. ~~(claim 3).~~ This arrangement has further advantages ~~further to those in claim 2~~ that it allows to have a relatively large annular gap, which allows cooling as well as centering of the rear axle with high precision.---

Please substitute the following amended paragraph for the first full paragraph on page 22 of the specification as originally filed:

---With the structure that the disk body has the supplying/discharging through holes open at its circular surface and the gate member is associated therewith for supplying/discharging of fluid, communication between the variable capacity compartments and the supplying/discharging channels is controlled in respect of predetermined angular positions of the gate member. ~~(claim 4).~~ ---

Please substitute the following amended paragraph for the second full paragraph on page 22 of the specification as originally filed:

---This arrangement has further advantages ~~further to those in claim 1~~ that it allows effective supplying/discharging of fluid with very little loss by controlling the angular positions of the gate member to communicate with the supplying/discharging channels, and further allows a quiet operation.---